

1 **A deep learning approach to the texture optimization problem for friction**  
2 **control in lubricated contacts**

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13

14 **Abstract:** The possibility to control friction through surface micro texturing could offer invaluable  
15 advantages in many fields, from wear and pollution reduction in the transportation industry to improved  
16 adhesion and grip. Unfortunately, the texture optimization problem is very hard to solve using  
17 traditional experimental and numerical methods, due to the complexity of the texture configuration  
18 space. In this work, we apply machine learning techniques to perform the texture optimization, by  
19 training a deep neural network to predict, with extremely high accuracy and speed, the Stribeck curve  
20 of a textured surface in lubricated contact. The deep neural network was used to completely resolve  
21 the mapping between textures and Stribeck curves, enabling a simple method to solve the texture  
22 optimization problem. This work demonstrates the potential of machine learning techniques in texture  
23 optimization for friction control in lubricated contacts.

24 **Keywords:** friction, surface texturing, optimization, deep learning, lubricated contact

## 25 **1 Introduction**

26 Our world is overwhelmed by the environmental impact of human activity and there is an imperative  
27 need to reduce pollution and mitigate its effects to avoid an irreversible global warming. The  
28 transportation industry, one of the largest contributors to polluting emissions, wastes a significant part  
29 of fuel and energy in overcoming friction forces between moving parts in contact [1] , meaning that  
30 any solution to reduce friction would provide huge environmental and economic benefits. Because of  
31 this, research on friction reduction has always been at the forefront of tribology research and many  
32 possible solutions exist, such as the application of surface coatings [2] and the use of more performing  
33 and environmentally friendly lubricant formulations [3]. One of the most promising ways to control  
34 the friction between contacting surfaces is provided by surface texturing, a process that is increasingly  
35 more efficient due to significant processing advances [4] allowing for rapid generation of patterned  
36 surfaces. It is well known that a fine control of friction through surface texturing can be achieved in  
37 nature. For example, sharks are covered in a regular array of denticles which help to achieve drag  
38 reduction [5]. The same reduction has been seen in the skin of snakes and certain lizards that developed  
39 scales to reduce dry contact friction [6]. Specific nano-hierarchically structured patterns found in the  
40 feet of tree toads [7], [8] and geckos [9] have been shown to provide a strong boundary friction,  
41 granting them better grip on vertical surfaces. In engineering applications, many different kinds of  
42 nature-inspired patterns have also been tested for friction control [8].

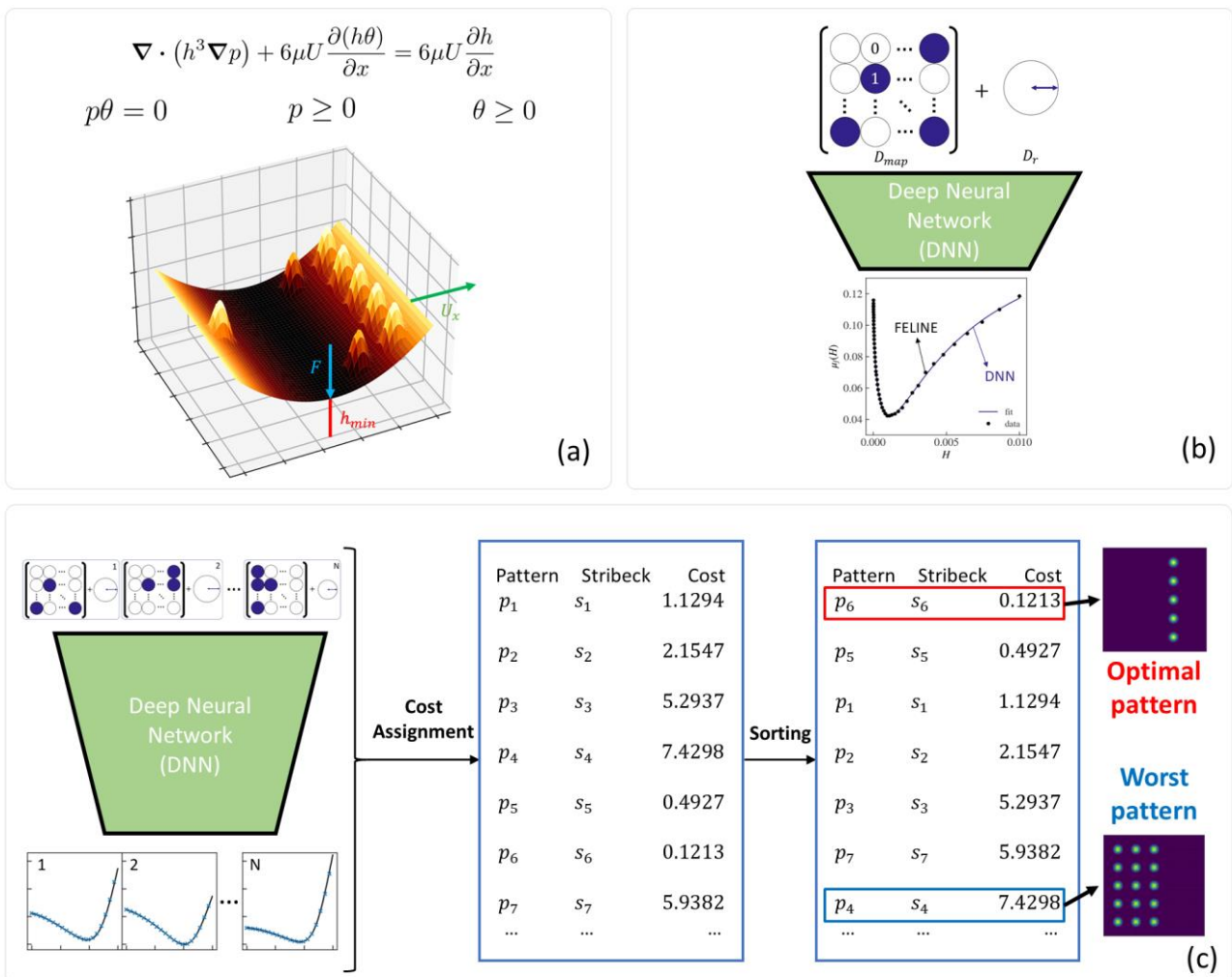
43 However, the design of these textures is in general based on trial-and-error methods, meaning that  
44 the optimal texture for a specific application is extremely hard to find. From an experimental  
45 perspective, textured samples need to be fabricated and tested, thus optimizing a specific pattern would  
46 require an extensive sampling of the texture parameter space, resulting in time and resource costs that  
47 are prohibitive [10], [11]. The same problem occurs when using numerical approaches to evaluate the  
48 tribological performance of a system, where the Stribeck curve [12] is calculated by solving the  
49 Reynolds equation [13], [14], for multiple sliding speeds, coupled with a model for treating the contact

50 friction [15]. Even if the simulation process is faster as a whole when compared to a single experiment,  
51 the calculations still require typically minutes to complete, meaning that our ability to sample the  
52 possible configuration space is incredibly limited [16]. Moreover, the relationship between patterns  
53 and resulting Stribeck curves is expected to be highly non-linear, based on current experimental and  
54 numerical understanding [4], [17]–[19]. A possible solution to the apparently insurmountable texture  
55 optimization problem might be offered by machine learning techniques. Machine learning (ML)  
56 encompasses a large range of algorithms and modeling tools used for large data processing tasks [20],  
57 [21] with typical applications being classification and regression problems in information technology  
58 [22], [23]. One of the most prominent ML techniques is represented by deep neural networks (DNN),  
59 which are used with considerable success in many fields of physics, from applications in condensed  
60 matter [24], [25] and materials science [25], [26] to the solution of complex nonlinear equations [27],  
61 [28].

62 Tribo-informatics has recently emerged as a new research area combining tribology with big data  
63 methods such as machine learning and artificial intelligence techniques [29]–[31]. These approaches  
64 can help in establishing new correlations in tribological data to predict the behavior of novel materials,  
65 provide novel insights and a broader understanding of the friction and wear mechanisms [32]. A recent  
66 example is the development by Almqvist of a physics-informed neural network (PINN) capable of  
67 solving the one-dimensional Reynolds equation [33]. This approach is meshless, thus it solves one of  
68 the main bottlenecks in traditional numerical solutions of lubricated contacts, that is the reliance on a  
69 mesh. What makes DNNs particularly appealing for the texture optimization problem is their universal  
70 approximation capability [22], [23], coupled with their extreme speed when compared to traditional  
71 methods [28]. In texture optimization problems, DNNs have been used to optimize the features of  
72 periodic patterns of nanopillars in optic metamaterials to achieve the desired properties, i.e., high  
73 electromagnetic wave absorption in some frequency windows [28]. In these works, a DNN replaced

74 the Maxwell equations solver, and it could predict -in millisecond time- an absorbance spectrum based  
 75 solely on the periodic pattern features.

76 In this work we developed an effective method for the optimization of surface texturing patterns for  
 77 friction applications based on a deep neural network. The DNN was designed and trained to accurately  
 78 predict the Stribeck curve of a dimple textured surface, thus replacing the standard Reynolds equation  
 79 solver in the solution of the forward problem. Moreover, to solve the inverse problem, a fast search-  
 80 based approach was implemented to predict a set of candidate surface parameters (dimple pattern and  
 81 dimple radius) that yield a set of closely matching Stribeck curves. The performance and accuracy of  
 82 the DNN and the inverse approach were validated by comparing with the solutions provided by a  
 83 numerical solver of the Reynolds and contact friction model equations.



84

85 **Figure 1.** Schematic representation of the implementation of the DNN solution for the forward and  
86 inverse problem in texture optimization. (a) Non-conformal contact of surfaces subject to load  $F$   
87 moving relative to each other with speed  $U$ , modeled as a height profile function  $\mathbf{h}(\mathbf{x}, \mathbf{z})$ . (b) Machine  
88 learning approach to predict the Stribeck curve of a textured surface, defined as the forward problem.  
89 The texture has 25 possible dimples (in a 5x5 grid with their presence represented in binary) and a  
90 fixed dimple radius for every dimple, therefore 26 parameters are capable of fully describing the  
91 texture. The output is a set of 7 parameters that allows for the reconstruction of the Stribeck curve. (c)  
92 Machine learning approach to solve the texture optimization problem (inverse problem). The Stribeck  
93 curves of the full configuration space are obtained by using the forward DNN. A cost value is then  
94 assigned to each pattern/Stribeck pair and a sorting algorithm is applied to obtain the extreme cases.

95

## 96 **2 Methods**

### 97 **2.1 Model for lubricated non-conformal contact of textured surfaces**

98 We consider a parabolic shape fully lubricated non-conformal contact of total area  $A$  similar to a  
99 lubricated journal bearing. Let  $h_0 \equiv h_0(x, y)$  describe the untextured gap geometry (height profile)  
100 between the two surfaces moving in relative motion with speed  $U$  while subject to an external load  $F$ ,  
101 as it is schematically represented in Fig. 1(a). The texturing is introduced by creating a 5x5 grid of  
102 cosine square shaped dimples. The dimple profile heights  $h_d$  are added to the untextured profile so that  
103 the textured gap geometry is defined as

$$h \equiv h_0(x, y) + h_d(x, y) \quad (1)$$

104 where  $h$  is the total height profile,  $h_0$  is the untextured gap geometry height profile and  $h_d$  is the  
105 dimple textured gap geometry height profile (further details on the contact geometry can be found in  
106 SI, Section I).

107 To obtain the pressure profile  $p$  within the lubricant we solve the Reynolds equation, derived from  
108 the Navier-Stokes equations [14] after considering the lubricant film to be at constant temperature  $T$ ,

109 constant density  $\rho$ , and constant viscosity  $\mu$ . Additionally, we do not consider any surface deformation  
 110 effects, therefore the shape of the height profile remains constant throughout the simulations. The  
 111 influence of the divergent domain on the film density is the possible formation of film rupture caused  
 112 by cavitation (formation of vapor filled cavities) [34]–[37]. To consider this effect we introduce a  
 113 system of equations for the pressure  $p$  and cavitation fraction  $\theta$  profiles:

$$\nabla \cdot (h^3 \nabla p) + 6\mu U \frac{\partial(h\theta)}{\partial x} = 6\mu U \frac{\partial h}{\partial x} \quad (2)$$

$$p\theta = 0 \quad (3)$$

$$p \geq 0 \quad (4)$$

$$\theta \geq 0 \quad (5)$$

114 The system of Eqs. (1)-(4) represents a linear complementarity problem (LCP) [38]–[40], which  
 115 was solved using the inexact Newton (INE) method [41] by restructuring the system of equations into  
 116 a damped Newton iteration. The INE method ensures that the solution follows the non-negativity  
 117 conditions at every iteration, thus providing a correct physical description of cavitation boundaries  
 118 [16].

119 Depending on sliding speed, applied load and viscosity, the system admits three different regimes  
 120 of lubricated contact: boundary, mixed and hydrodynamic, which differ in their main friction  
 121 mechanisms. In this work only the mixed and hydrodynamic regimes have been considered since they  
 122 occur in the presence of lubricant within the contact whereas in the boundary regime the surfaces are  
 123 in direct contact (dry friction). To treat the mixed contact regime, both the hydrodynamic and asperity  
 124 contact forces need to be considered in the same model. To this end, we decided to adopt a load-sharing  
 125 approach [42], where the total friction force results from the combination of the hydrodynamic and  
 126 contact terms:

127

$$f_t = f_{hydro} + f_{GT}. \quad (6)$$

128 The first term, valid only in regions where film rupture does not occur, represents the hydrodynamic  
 129 component of friction, and depends on the pressure gradient generated within the lubricant film,

$$f_{hydro} = \left( \frac{h}{2} \frac{\partial p}{\partial x} + \frac{U\mu}{h} \right) A_f, \quad (7)$$

130 where  $A_f$  is the area of the full film domain. The second term of Eq. (5) represents the contact  
 131 component of friction, that was calculated using the Greenwood-Tripp (GT) contact model [15, 42],  
 132 which considers the roughness of both contacting surfaces. The GT contact model was chosen because  
 133 it is simpler to implement while providing a qualitative treatment of friction [43]. Clearly, the GT  
 134 contact model is not the best choice, as it is known to underestimate the pressure [44]–[46] and more  
 135 accurate choices exist [47]. However, the focus of this work is to demonstrate the applicability of a  
 136 DNN to solve the optimization problem of textured surfaces in lubricated contact, which does not  
 137 depend on the particular choice of the solver used to generate training data.

138 In the GT model, the total load carried is defined as

$$f_{GT} = \tau_0 A_a + \mu_{asp} W_{asp}, \quad (7)$$

139 where  $\tau_0$  is the Eyring shear stress,  $A_a$  is the asperity contact area,  $W_{asp}$  is the load carried by  
 140 asperities in the contacting surfaces,  $\mu_{asp}$  is the coefficient of friction of the asperities [43], [48]. The  
 141 load carried by asperities is defined as

$$W_{asp} = \frac{16\sqrt{2}}{5} \pi (\eta k \sigma)^2 \sqrt{\frac{\sigma}{k}} A E' F_{5/2} \left( \frac{h}{\sigma} \right). \quad (8)$$

142 The asperity contact area is similarly defined as

$$A_a = \pi^2 (\eta k \sigma)^2 A F_2 \left( \frac{h}{\sigma} \right), \quad (9)$$

143 The functions  $F_{n/m}(h/\sigma)$  are statistical functions that account for the Gaussian distribution of  
 144 asperities and can be approximated through a parametric fit [49]. The chosen values of the parameters  
 145  $\eta$ ,  $k$ , and  $\sigma$  present in Eqs. (8)-(9) reflect the typical values for a well-polished steel surface [15]. It

146 should be noted that this roughness scale is much smaller than dimple characteristic dimensions. A  
147 table of numerical values for the parameters used in data generation in the training of the deep neural  
148 network is available in Section 3 of SI.

149 An open-source finite element implementation of the solver for the system of Eqs. (1)-(6), FELINE  
150 [50], was specifically developed and used to generate the training data and validate DNN results.

151

152

## 153 **2.2 Method of solution and Stribeck curve calculation**

154 To obtain the Stribeck curve, one must compute the coefficient of friction (COF) for each relevant  
155 sliding speed as the surface integral of the total friction force in (5), written as:

$$COF(H) = \frac{1}{WA} \int_{\Omega} f_t(H) d\Omega, \quad (10)$$

156 where

$$H = \frac{\mu U}{F} \quad (11)$$

157 is a dimensionless parameter dependent of the relative sliding speed termed Hersey number, and  $W$  is  
158 the total load carried, including both the hydrodynamic and contact term. A total of 50 different Hersey  
159 number values, equally spaced in a logarithmic scale, were used for each Stribeck curve in the interval  
160  $H \in [10^{-5}, 10^{-2}]$ . This is equivalent to a procedure where the relative sliding speed is changed in the  
161 interval  $U \in [0.043, 43] m/s$ .

162 For every value of  $H$ , the COF is calculated when the total load carried balances the applied load  $F$ .  
163 For this purpose, we first perform a solver run with an initial condition for the minimum separation  
164  $h_{min}$ . The total carried load  $W$  is then calculated and compared with  $F$ , and the minimum separation  $h_{min}$   
165 is adjusted by lowering (raising) it if  $W$  is smaller (larger) than  $F$ . After three iterations a spline  
166 interpolation is used to accelerate convergence, which is reached when  $F^{-1}(W - F) < 10^{-3}$ .

167



### 168 2.3 Design and training of the DNN

169 The textured surface in a lubricated contact is defined by a set of parameters: the dimple map  $D_{map}$   
170 of dimension  $(D_x, D_y)$  which describes the presence and position of dimples on the surface (see  
171 Section 1 of S.I.), the dimple depth  $D_d$ , the dimple radius  $D_r$ , the parabolical edge  $E_0$ , and the surface  
172 roughness parameters  $\eta k \sigma$ . The value of dimple depth was fixed to  $D_d^0 = 6 \mu m$  to isolate the influence  
173 of dimple radius. This way, the complexity of the optimization problem is greatly reduced, with only  
174 one tunable parameter present in addition to the dimple map. Further details regarding the effect of  $D_r$   
175 and  $D_d$  on load carrying capacity (LCC) and  $h_{min}$  of textured contacts can be found in SI.

176 Due to the fact that we considered a  $5 \times 5$  grid of dimples with 6 possible  $D_r$  values in the interval  
177  $[40, 60] \mu m$ , our network input consists of 26 parameters, that is 25 possible spots for dimple  
178 placement represented as Boolean variables and a globally applied value for dimple radius. The  $D_r$   
179 interval was selected since it provides a sufficient range for optimization while remaining within the  
180 validity conditions of the Reynolds equation. Even if such configuration space appears simple at a first  
181 glance, it contains a total of  $N = 6 \times 2^{25} \approx 2.01 \times 10^8$  possible texture configurations, rendering the  
182 texture optimization problem impossible to solve for any traditional solution approach.

183 In order to represent all Stribeck curves in the configuration space with the same number of  
184 parameters, we performed a fit of our data calculated with FELINE using a rational polynomial form  
185 defined as

$$f_n^m(x) = \frac{p_1 x^n + p_2 x^{n-1} + \dots + p_n x + p_{n-1}}{x^m + q_1 x^{m-1} + \dots + q_{m-1} x + q_m}, \quad (12)$$

186 with polynomial degrees  $(n, m) = (3, 3)$ . This rational polynomial was found to be the best  
187 compromise between accuracy and total number of parameters when representing a Stribeck curve.

188 As a result, the DNN output consists of 7 rational fit parameters that allow the reconstruction of the  
189 Stribeck curve, thus reducing the overall number of output parameters while not significantly affecting  
190 precision. This fitting step regularizes the output of the network.

191 Regarding the DNN architecture, we adopted a simple topology consisting of 6 fully connected  
192 hidden layers with a number of neurons {32, 64, 96, 96, 64, 32} under no regularization using the  
193 ReLU activation function with He normal initialization [51] due to its performance and simplicity [52],  
194 [53]. Since the number of hidden layers in our network is small and the width of these layers is  
195 sufficiently large, we do not expect the "dying ReLU problem" in this study [54]. The optimizer of  
196 choice was Nadam as it incorporates Nesterov momentum (with default hyperparameters) and can  
197 improve the convergence of the learning process [55]–[57]. To further optimize the training process,  
198 we sampled the validation set loss as a function of learning rate after 10 steps of training. This allowed  
199 us to carefully select a value for the learning rate ( $10^{-3}$ ). The RMSE of the predicted rational fit  
200 coefficients was used as the network loss function. The training and testing process was implemented  
201 with Python 3.9 using the Keras high-level API [58] of TensorFlow version 2 [59].

202 The DNN training set was populated by randomly sampling sets of dimple maps  $\mathbf{D}_{map}$ , whose  
203 corresponding patterns were solved for all the  $D_r$  values to obtain the corresponding Stribeck curves.  
204 In total, around 60000 different combinations of patterns and dimple radii were computed using the  
205 FELINE solver, which required 6 days of computation time on 300 simultaneously running processes  
206 on Intel(R) Xeon(R) CPU E5-2697 v2 cores. This dataset was standardized according to the  
207 StandardScaler utility in the scikit-learn python package [60].

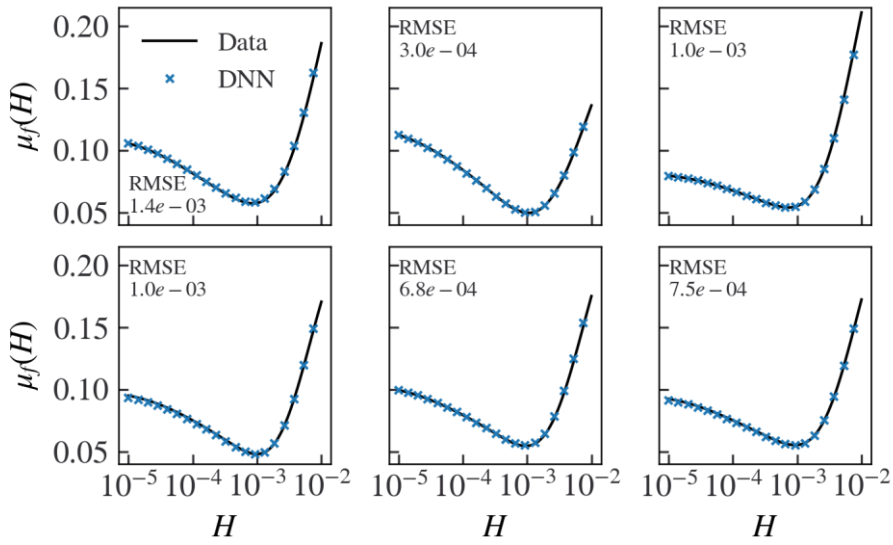
208 Owing to the fact that the boundary pressure is the same at  $y = 0$  and  $y = L$ , we expect that a mirror  
209 reflection of any pattern over the  $y = L/2$  axis does not change the corresponding Stribeck curve. This  
210 symmetry was explicitly included in the dataset by assigning the same Stribeck curve both to a pattern  
211 and to its reflection. This step is important in enhancing the overall physical accuracy of the DNN,  
212 while requiring no additional generation of data. From the generated dataset we selected 10% as a  
213 validation set, thus our resulting training set contains 54000 pairs of surface parameters and Stribeck  
214 curves and accounts for only 0.05% of the total configuration space.

215

216 **3 Results and discussion**

217 **3.1 Solution of the forward problem**

218 The forward problem, schematically represented in Fig. 1(b), was solved and examples of the DNN  
 219 predictions are shown in Fig. 2 for a few cases in the validation set compared to data produced with  
 220 the FELINE solver. The median RMSE of cases in the validation set is  $5.7 \times 10^{-4}$ , meaning that the  
 221 network predictions are very accurate and show no appreciable difference with the Stribeck curves  
 222 calculated with FELINE.



223  
 224 **Figure 2.** Network prediction of the Stribeck curves for randomly selected patterns in the validation  
 225 set. The prediction accuracy is evaluated in terms of the root mean squared error (RMSE) in  
 226 comparison to true data. The DNN results (crosses) have been down sampled for clarity.

227  
 228 Fig. 3(a) shows a histogram of the RMSE distribution for the validation set predictions of the full  
 229 Stribeck curve and, for two separate regimes of the Stribeck curve, that is the mixed regime and the  
 230 hydrodynamic regime. The same analysis was performed on a test set of patterns (~4000 samples) and  
 231 it was verified that the network performs well on an arbitrary set of patterns (see SI, section X). To  
 232 correctly establish boundaries for these regimes we used the lambda parameter criteria [61]:

$$\lambda = \frac{h_{min}}{\sigma}, \quad (83)$$

233 where  $h_{min}$  denotes the minimum thickness of the lubricant film (or minimum distance between the  
 234 contacting films) and  $\sigma$  is one of the surface roughness parameters. For  $\lambda > 3$  the contact regime is  
 235 said to be hydrodynamic, while the mixed regime occurs for  $1 < \lambda < 3$ . After taking an average of  $\lambda$   
 236 for all curves in the validation set we found that the averaged value  $H = 0.0015$  represents well the  
 237 point in which the lubrication regime changes. Therefore, for  $H \in [0, 0.0015]$  we have the mixed  
 238 regime and for  $H \in [0.0015, 0.01]$  we have the hydrodynamic regime.

239 The corresponding median, 95th and 99th percentile of the different histograms is reported in  
 240 Table 1. Low RMSE values ( $< 10^{-3}$ ) are consistently encountered in all regimes, indicating that the  
 241 trained DNN is reliable across all the data. However, a better accuracy of the DNN in the mixed region  
 242 was observed, compared to the hydrodynamic region. This is likely due to the larger span of COF  
 243 values in the hydrodynamic region, for the same number of training samples, resulting in a lower  
 244 accuracy prediction of the DNN therein.

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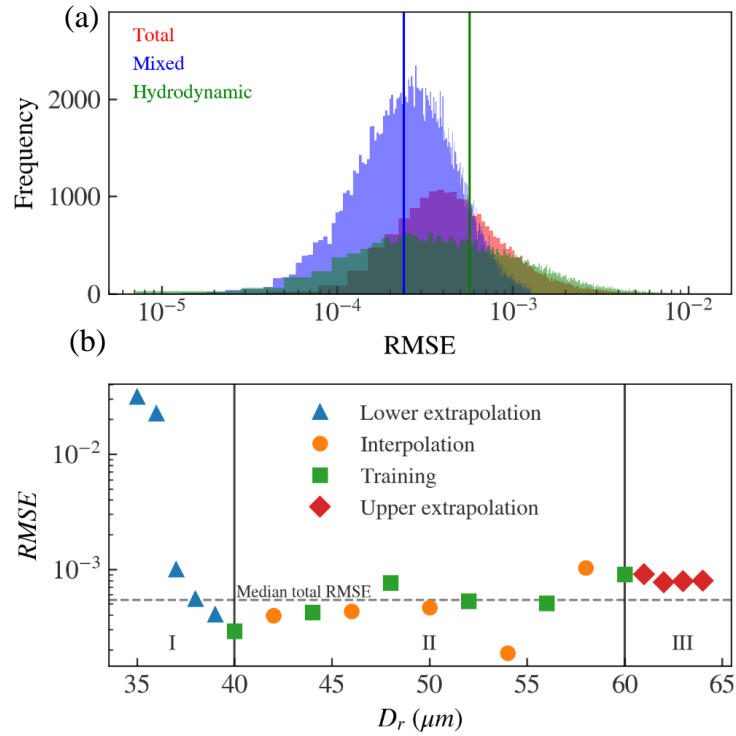
246 **Table 1** Median, 95th and 99th percentile of the RMSE values shown in the histogram in Fig. 3(b).

Regime	Median	99%	95%
Mixed	$2.4 \times 10^{-4}$	$7.9 \times 10^{-4}$	$6.0 \times 10^{-4}$
Hydrodynamic	$9.4 \times 10^{-4}$	$6.1 \times 10^{-3}$	$3.8 \times 10^{-3}$
Total	$5.7 \times 10^{-4}$	$3.4 \times 10^{-3}$	$2.1 \times 10^{-3}$

247

248 To assess the quality of the trained network, it is also important to verify its ability to interpolate  
 249 and extrapolate results in terms of the dimple radius, since it was trained only with 6 possibilities for  
 250 it. In this regard, we compared both the network and the FELINE solver solutions for values in-between  
 251 and outside the dimple radius interval  $40 \mu\text{m}$  to  $60 \mu\text{m}$ .

252 A pattern was randomly picked and its corresponding Stribeck curve was computed with the DNN  
 253 and FELINE in order to obtain a RMSE of their difference, which was plotted in Fig. 3(c) as a function  
 254 of  $D_r$ . As one can see in Fig. 3(c), in region (II), the interpolation region, there is a small difference  
 255 between the interpolation results from the DNN and the corresponding ones from the FELINE solver,  
 256 showing that the network is capable of accurate interpolating behavior.



257  
 258 **Figure 3.** (a) Histogram of the RMSE for all patterns in the validation set in different regimes. The  
 259 medians for the mixed, hydrodynamic, and total regions are also shown. (b) Interpolation and  
 260 extrapolation study of dimple radius versus RMSE where: (I) indicates the lower extrapolation bound  
 261  $D_r \in [35, 39]$ , (II) the interpolated values  $D_r \in [42, 46, 50, 54, 58]$  and the values used in training,  
 262 (III) the upper extrapolation bound  $D_r \in [61, 65]$ .

263  
 264 For the extrapolation cases, regions (II) and (III), we see that the lower extrapolation bound works  
 265 significantly worse than the upper extrapolation bound. In the hydrodynamic regime of lubrication, the  
 266 COF increases linearly with increasing Hersey number. Contrary to this, in the mixed regime of

267 lubrication, the COF increases exponentially with decreasing Hersey number. Extrapolation is  
268 typically more accurate for linear behavior, hence resulting in a larger extrapolation error for the lower  
269 bound of extrapolation.

270 In terms of timing, the DNN is  $10^6$  times faster when compared to the FELINE solver. In  
271 conclusion, we have successfully designed a DNN that meets the requirements of speed and accuracy  
272 needed to fully solve the texture optimization problem for tribological applications.

273

### 274 **3.2 Solution of the inverse problem**

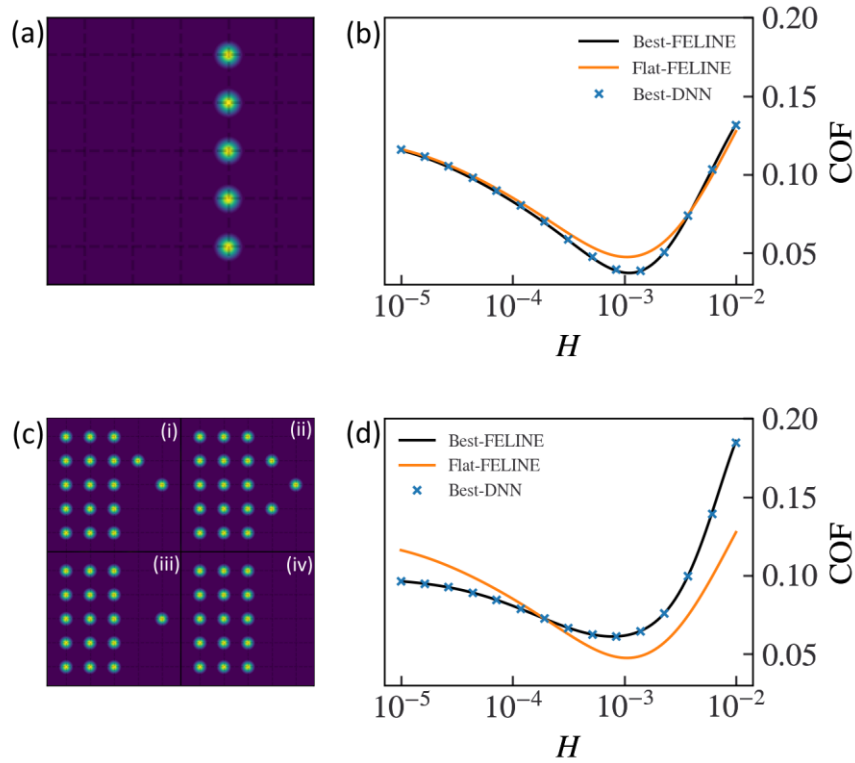
275 The inverse problem, schematically represented in Fig. 1(c), consists in the ability to predict optimal  
276 patterns starting from desired characteristics of a particular system. With our capable forward problem  
277 DNN solver it is possible to obtain the Stribeck curve for every case in the full configuration space  
278 defined by 25 possible dimples and 6 possible radii, that is a total of about  $3.3 \times 10^8$  cases. Let  
279  $(p_k, s_k)$  stand for the k-th Pattern/Stribeck pair in the configuration space and consider a value, or cost,  
280  $C \equiv C(p_k, s_k)$  that we can assign to each pair, such as the minimum of the Stribeck curve  $s_k$ . By  
281 sorting the list of possible cases by their cost, we obtain the cases with smallest COF minimum and  
282 also highest COF minimum at the top and bottom of the list, respectively. The simple and  
283 straightforward algorithm Quicksort [62] was employed for this task.

284 The resulting optimal pattern for obtaining the smallest COF minimum using this method is shown  
285 in Fig. 4(a) and the corresponding Stribeck curve (solved with both FELINE and the DNN) is also  
286 reported in Fig. 4(b), compared with the untextured case. We observe that in this case the dimples are  
287 present only in a small region right after  $h_{min}$ . The pressure and cavitation profiles of the contact  
288 obtained using FELINE, for the predicted optimal texture at the lowest friction point in the Stribeck  
289 curve are reported in Fig. 5(a) and 5(b), respectively. We observed an increase in pressure in the  
290 convergent gap and also a smaller pressure increase in the divergent gap. Furthermore, a significant  
291 drop in the cavitation profile led to an overall reduction of the size of the cavitation region in the

292 divergent gap. The presence of dimples right after the  $h_{min}$  zone appears to hinder the formation of  
 293 the cavitation region. Thus, the optimal pattern results in smaller film rupture areas and larger  
 294 pressures in the convergent area.

295 The configuration space sorting also allows us to find the pattern that yields the largest COF  
 296 minimum. We found a family of very similar patterns that generated almost indistinguishable Stribeck  
 297 curves, as shown in Fig. 4(c)-(d). In this case, the convergent region of the contact is fully textured,  
 298 including the  $h_{min}$  area. By looking at the pressure profile (Fig. 5(c)) we observed a significant  
 299 pressure drop in the convergent gap. This is caused by the presence of dimples in a region that produces  
 300 most of the hydrostatic pressure that counters the applied load. Such dimple placement disrupts the  
 301 pressure profile. The cavitation profile (Fig. 5(d)) also displayed a decrease in magnitude, but the total  
 302 cavitation area was not affected by this.

303



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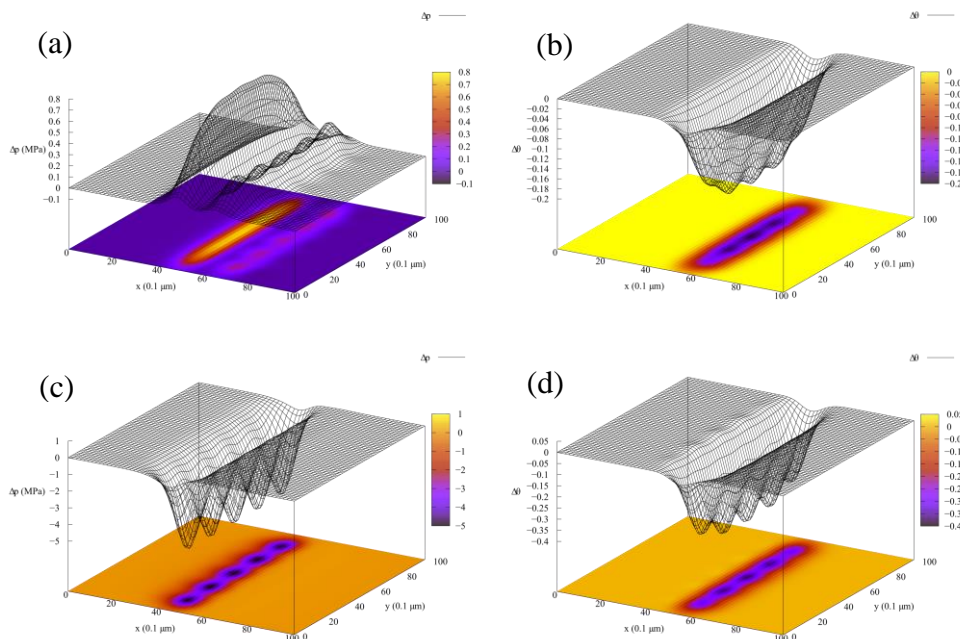
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306

307 **Figure 4.** Resulting patterns and Stribeck curves from the cost assignment and sorting method used to  
 308 find the optimal friction reducing/increasing patterns in the mixed regime. (a) Optimal pattern that  
 309 yields the Stribeck curve with the smallest minimum. The corresponding Stribeck curve, calculated  
 310 with both FELINE and the DNN, is reported in panel (b), where it is compared with the untextured  
 311 case. (c) Family of textures that yield nearly matching Stribeck curves with the largest COF minimum.  
 312 The Stribeck curve of pattern (i) calculated with both FELINE and the DNN is reported in panel (d),  
 313 where it is compared with the untextured case. The DNN results (crosses) have been down sampled  
 314 for clarity.

315

316 Curiously, the two families of patterns that provide a significant decrease (Fig. 4(a)) and increase  
 317 (Fig. 4(c)) in COF, have been already observed and described by Tala and collaborators [63]. The  
 318 reasoning behind the friction increase/decrease phenomena described in Tala et. al. accurately matches  
 319 the observations of our own neural network predictions and further investigation done with FELINE.  
 320 This fact demonstrates that our solution of the inverse problem is capable of correctly identifying the  
 321 optimal texturing pattern. In terms of accuracy, the DNN solutions are basically indiscernible from the  
 322 FELINE ones.



323



324 **Figure 5.** Pressure and cavitation profiles, determined at the Hersey number of the COF minimum of  
 325 the untextured case Stribeck curve. For the optimal pattern in Fig. 4(a) for friction reduction and for  
 326 friction gain in Fig. 4(c). (a) Pressure difference between the untextured and textured case in Fig. 4(a).  
 327 (b) Cavitation difference between the untextured and textured case in Fig. 4(a). (c) Pressure difference  
 328 between the untextured and textured case in Fig. 4(c). (d) Cavitation difference between the untextured  
 329 and textured case in Fig. 4(c).

330

331 The collection of similar patterns (i)-(iv) shown in Fig. 4(c) exemplifies an important feature of a  
 332 method like this, which is the ability to find nearly identical solutions generated by different patterns.  
 333 This can be extremely important from an experimental point of view because, at parity of performance,  
 334 a particular texture may be better in terms of manufacturing cost/speed in a laboratory.

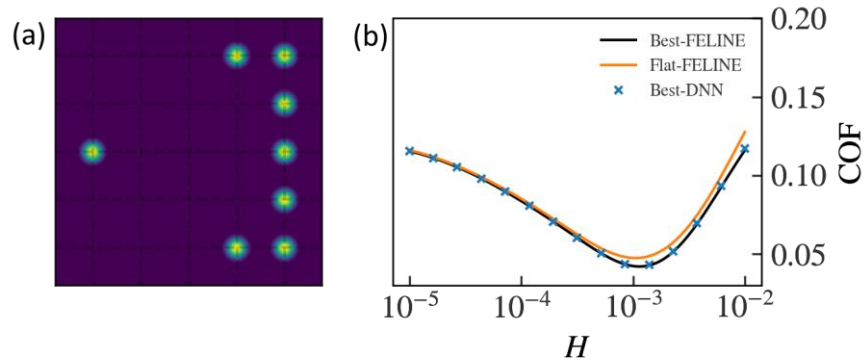
335 To further test our inverse problem solver, we looked for a pattern that reduces friction in the  
 336 hydrodynamic regime. To achieve this, we defined a more elaborate form of the cost:

$$C(p_k, s_k) = \int_{H_1}^{H_2} [s_k(H) - s_0(H)] dH, \quad (9)$$

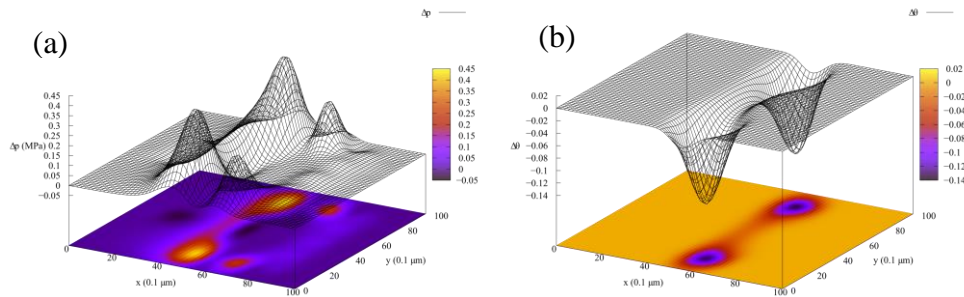
337 where  $H_1, H_2$  encompass the hydrodynamic range. This function computes the area under the Stribeck  
 338 curve in the hydrodynamic region for some Stribeck curve  $s_k \in L$  minus the area under the Stribeck  
 339 curve of the untextured case,  $s_0$ . The lowest value of  $C$  should correspond with the pattern that  
 340 maximizes the reduction of the COF in the hydrodynamic regime.

341 The resulting optimal pattern for maximizing the reduction of the COF in the hydrodynamic  
 342 regime using this method is shown in Fig. 6(a), and the corresponding Stribeck curve (solved with  
 343 both FELINE and the DNN) is also reported in Fig. 6(b), compared with the untextured case. The  
 344 corresponding pressure and cavitation profile differences with the untextured cases are reported in  
 345 Fig. 7(a) and 7(b), respectively. It is possible to observe an increase of pressure in the convergent and

346 divergent gaps (Fig. 7(a)). Concerning cavitation, only a slight reduction of the size of the cavitation  
 347 region in the divergent gap was observed.



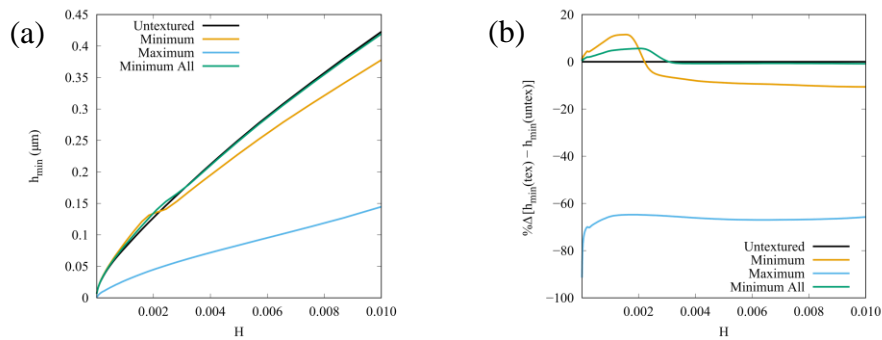
348  
 349 **Figure 6.** Resulting pattern and Stribeck curves from the cost assignment and sorting method used to  
 350 find the optimal friction reducing pattern in the hydrodynamic regime. (a) Optimal pattern that yields  
 351 the Stribeck curve with the overall smallest COF in the hydrodynamic regime. The corresponding  
 352 Stribeck curve, calculated with both FELINE and the DNN is reported in panel (b), where it is  
 353 compared with the untextured case. The DNN result (crosses) has been down sampled for clarity.



354  
 355 **Figure 7.** Pressure and cavitation profiles, determined at the Hersey number of the COF minimum of  
 356 the untextured case Stribeck curve. For the optimal pattern in Fig. 7(a) for friction reduction. (a)  
 357 Pressure difference between the untextured and textured case in Fig. 7(b). (b) Cavitation difference  
 358 between the untextured and textured case in Fig. 7(b).

359  
 360 The absolute values of  $h_{min}$  and the percent deviations of the difference of  $h_{min}$  relative to the  
 361 untextured case for the optimal texture cases for friction reduction/gain are represented, respectively,

362 in Fig. 8(a) and 8(b). For the optimal texture for friction reduction in the mixed regime (orange line in  
 363 Fig. 8), a ~12% increase in the  $h_{min}$  in the mixed regime and a ~10% decrease of  $h_{min}$  at larger H  
 364 numbers was observed. Additionally, for the mixed regime, calculations also show a ~5% increase in  
 365 the LCC coupled with a ~22% decrease in the overall cavitation region area, resulting in a smaller  
 366 COF for these sliding speeds. For the optimal texture for friction gain (cyan line in Fig. 8), an 80-90%  
 367 decrease in  $h_{min}$  in the mixed regime was observed, stabilizing around a 60% decrease at larger H  
 368 numbers. Concerning LCC and cavitation area, a 33% decrease and no change in the overall cavitation  
 369 region were found, explaining the significant increase in COF for this pattern. Finally, for the  
 370 optimal texture for friction reduction in all regimes (green line in Fig. 8), a ~5% increase in  $h_{min}$  in  
 371 the mixed regime was observed, while no appreciable variation could be seen at larger H numbers.  
 372 A ~3% increase in the LCC, coupled with a ~16% decrease in overall cavitation region area, shows a  
 373 similar performance compared to the Fig. 4(a) pattern. The combination of these effects results in a  
 374 slight decrease of COF for nearly all sliding speeds. Textures that are capable of an effective reduction  
 375 of friction and increase in LCC can highly contribute to extend the service life of the lubricated contact



376

377 **Figure 8.** Effect of texturing on the minimum film thickness as a function of the Hersey number. (a)  
 378 Comparison between the untextured case and the three cases determined by the neural network, where  
 379 Minimum is the texture of Fig. 4(a), Maximum is the texture of Fig. 4(c)(i) and Minimum All is the  
 380 texture of Fig. 6(a). (b) Percentual deviation of the difference of  $h_{min}$  with respect to the untextured  
 381 case.

382

383 The above optimal textures, and in particular the ones in Fig. 4(a) and Fig. 4(c), clearly demonstrate  
384 the capability of our approach for texture optimization problems. In fact, the solution of the inverse  
385 problem for the lowest and highest COF values returned the expected optimal textures [63]. The  
386 strength of this method is that the proposed solutions are actually the best ones, since all of the possible  
387 configuration space has been explored. This demonstrates how the texture optimization problem turns  
388 in to a very simple task using a DNN to solve forward problem, which allowed for a very efficient  
389 solution of texture configuration space.

390

#### 391 **4 Conclusions**

392 We have successfully designed and trained a deep neural network capable of accurately predicting  
393 the resulting Stribeck curve generated by a dimpled texture with median root mean square errors of  
394  $5.7 \times 10^{-4}$ . This type of texture, composed of an array of  $5 \times 5$  possible dimples with dimple radius  
395  $D_r$ , has an unpredictable and highly non-linear effect on the surface friction coefficient. The DNN can  
396 efficiently compute all possible cases of a total of around 300 million possibilities, trained with only  
397 0.05% of them, thus enabling us to solve the texture optimization problem which is otherwise  
398 impossible to treat by traditional experimental and numerical methods. We determined both extremes  
399 of an optimization problem by taking advantage of the incredible performance of our DNN, predicting  
400 the relevant optimal textures in the process. We investigated properties of the developed DNN such as  
401 accuracy, extrapolation, and interpolation capabilities, demonstrating its robustness and reliability.  
402 This work paves the way for the use of deep learning as a tool to realize careful friction control of  
403 surfaces through optimally designed textures.

404

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410

411

#### 412 **Declaration of competing interest**

413 The authors have no competing interests to declare that are relevant to the content of this article.

414

#### 415 **Electronic Supplementary Material (ESM)**

416 Electronic Supplementary Material: Supplementary material (add a brief description) is available in  
417 the online version of this article).

#### 418 **Data availability statement**

419 All the data used in this work is available free of charge from  
420 <https://doi.org/10.34622/datarepository/MUVOJD>

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